

# Subgrid-scale model for radiative transfer in turbulent participating media

Laurent Soucasse, Philippe Rivière, Anouar Soufiani

Laboratoire EM2C, CNRS, CentraleSupélec, Université Paris-Saclay  
laurent.soucasse@centralesupelec.fr

Journée SFT Rayonnement

22 novembre 2017

# Motivations

## **DNS of turbulent flows coupled with thermal radiation**

- Few coupled flow/radiation DNS in the literature
- Very fine spatial meshes, added to directional and spectral dependence for radiation

## **Radiation interactions**

- Radiation emission is local (can be computed on the DNS mesh)
- Radiation absorption is non local (contribution of all turbulent length scales)
- Radiation absorption associated with small turbulent length scales is only significant for high optically thicknesses

### **Objective**

Develop an accurate radiation model to be coupled with a flow DNS

## Motivations

### Radiative transfer model

- Filtering of the blackbody intensity
- Subgrid-scale model
- Implementation

### Results

- Homogeneous turbulent grey medium
- Natural convection in a differentially heated cavity
- Turbulent channel flow of hot combustion products

### Conclusion

# Radiative transfer model

## Filtering of the blackbody intensity

### Radiative transfer equation and BC

- Linear dependence with the Planck function

$$\mathbf{u} \cdot \nabla I_\nu(\mathbf{r}, \mathbf{u}) = \kappa_\nu(\mathbf{r}) [I_{b\nu}(\mathbf{r}) - I_\nu(\mathbf{r}, \mathbf{u})]$$

$$I_\nu(\mathbf{r}_w, \mathbf{u}) = \varepsilon_\nu(\mathbf{r}_w) I_{b\nu}(T_w(\mathbf{r}_w)) + \frac{1 - \varepsilon_\nu(\mathbf{r}_w)}{\pi} \int_{\mathbf{u}' \cdot \mathbf{n}_w < 0} I_\nu(\mathbf{r}_w, \mathbf{u}') |\mathbf{u}' \cdot \mathbf{n}_w| d\mathbf{u}'$$

### Filtering

$$I_{b\nu}(\mathbf{r}) = \overline{I_{b\nu}}(\mathbf{r}) + I'_{b\nu}(\mathbf{r})$$
$$I_\nu(\mathbf{r}, \mathbf{u}) = \tilde{I}_\nu(\mathbf{r}, \mathbf{u}) + I''_\nu(\mathbf{r}, \mathbf{u})$$

#### Filtered contributions

Deterministic ray-tracing method  
on a coarse mesh

#### Subgrid scales

Model in Fourier space

# Radiative transfer model

## Subgrid-scale model

The subgrid contribution is likely to be significant for high optical thicknesses. Therefore, two assumptions are made:

- (1) the absorption coefficient is considered uniform
- (2) the subgrid intensity leaving the boundaries is set to zero

### Subgrid model in Fourier space

- Fourier transform

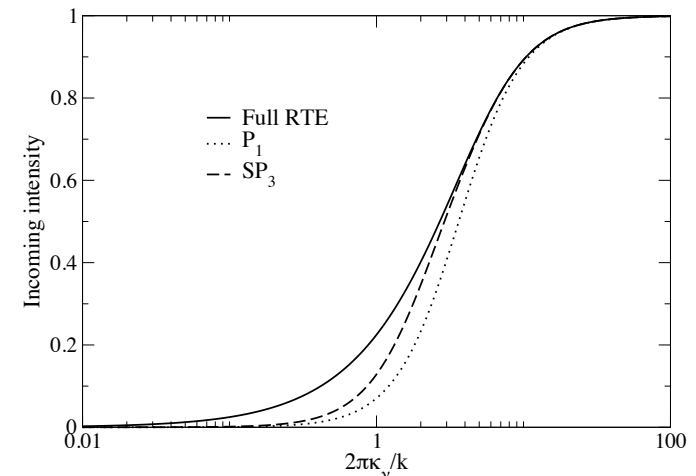
$$\hat{I}''_{\nu}(\mathbf{k}, \mathbf{u}) = \int_{\mathbf{r}} \exp(-i\mathbf{k} \cdot \mathbf{r}) I''_{\nu}(\mathbf{r}, \mathbf{u}) d\mathbf{r}$$

- RTE

$$(\kappa_{\nu} - i\mathbf{k} \cdot \mathbf{u}) \hat{I}''_{\nu}(\mathbf{k}, \mathbf{u}) = \kappa_{\nu} \hat{I}'_{b\nu}(\mathbf{k})$$

- Analytic solution

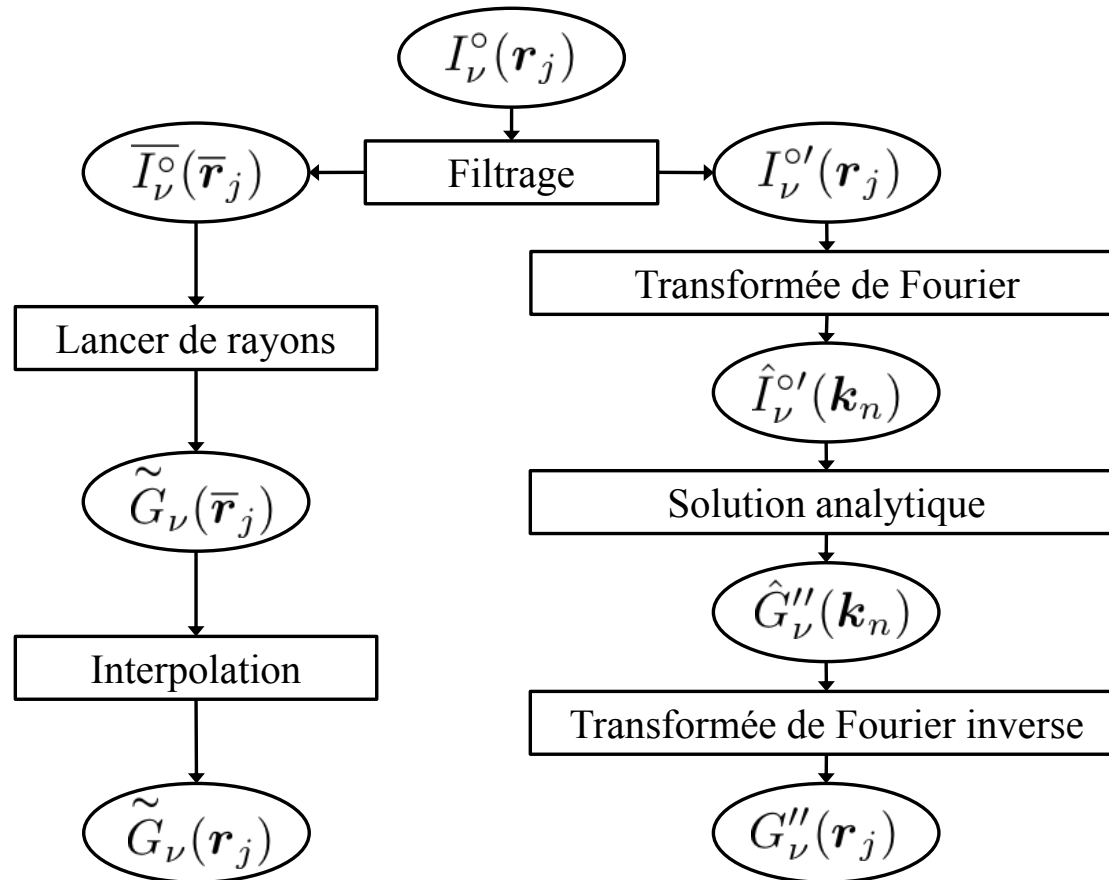
$$\begin{aligned} \hat{G}''_{\nu}(\mathbf{k}) &\equiv \int_{4\pi} \hat{I}''_{\nu}(\mathbf{k}, \mathbf{u}) d\mathbf{u} = \kappa_{\nu} \hat{I}'_{b\nu}(\mathbf{k}) \int_{4\pi} \frac{d\mathbf{u}}{\kappa_{\nu} - i\mathbf{k} \cdot \mathbf{u}} \\ &= 4\pi \hat{I}'_{b\nu}(\mathbf{k}) \left( \frac{\kappa_{\nu}}{k} \right) \arctan \left( \frac{k}{\kappa_{\nu}} \right) \end{aligned}$$



Subgrid intensity vs optical thickness of spatial fluctuations

# Radiative transfer model

## Implementation



$$P_\nu(\mathbf{r}) = \kappa_\nu(\mathbf{r})4\pi I_{b\nu}(\mathbf{r}) - \kappa_\nu(\mathbf{r})(\tilde{G}_\nu(\mathbf{r}) + G''_\nu(\mathbf{r}))$$

⇒ Gain en temps de calcul de l'ordre de  $(N/\bar{N})^3$

## Motivations

### Radiative transfer model

- Filtering of the blackbody intensity

- Subgrid-scale model

- Implementation

### Results

- Homogeneous turbulent grey medium

- Natural convection in a differentially heated cavity

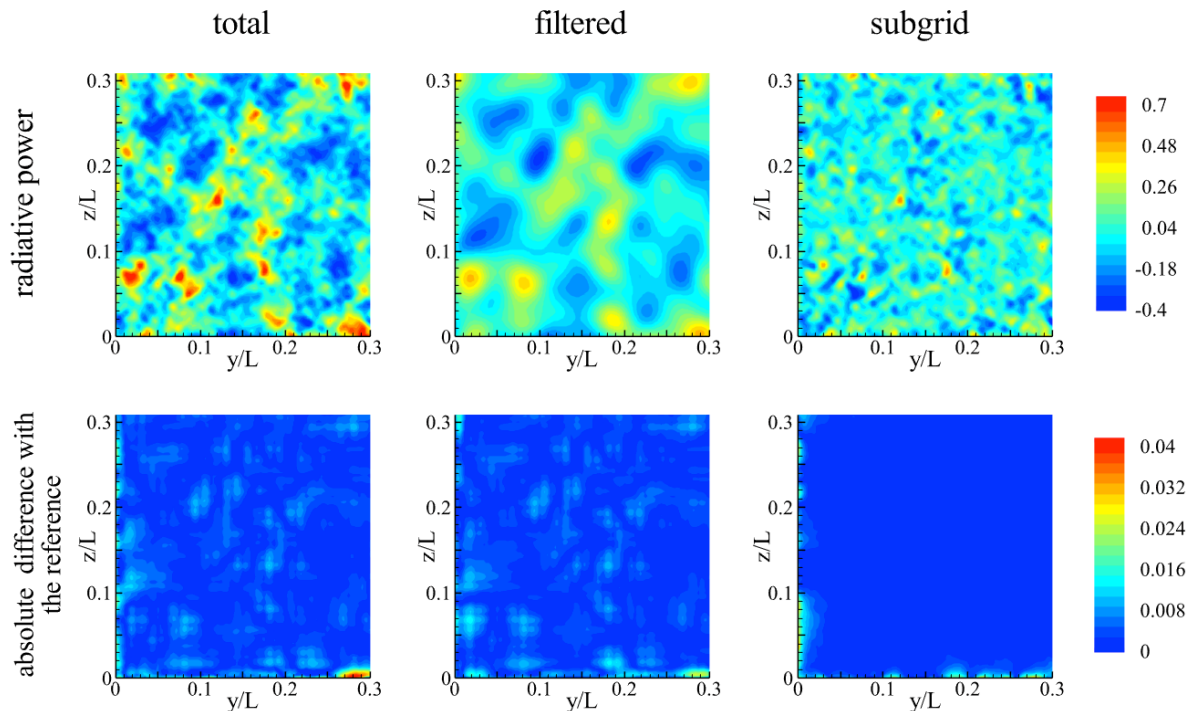
- Turbulent channel flow of hot combustion products

### Conclusion

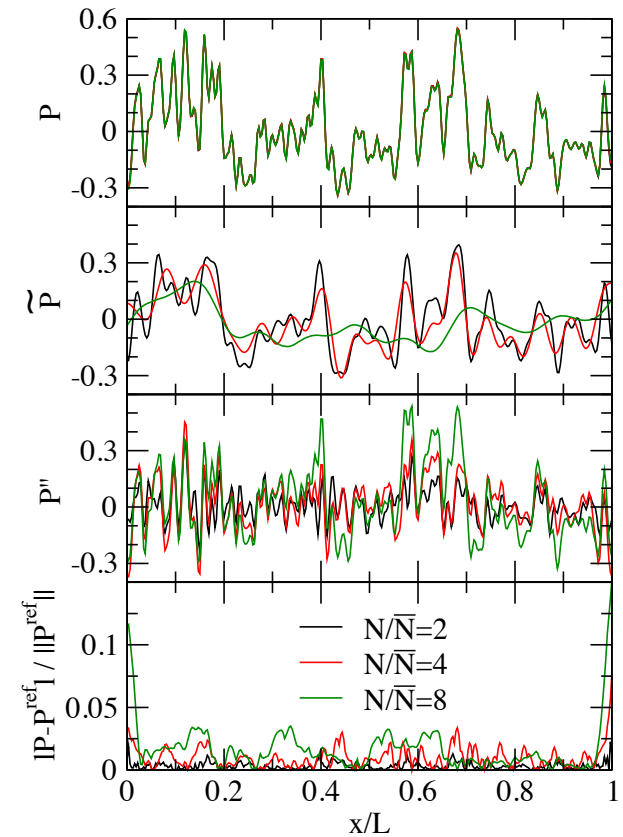
# Results

## Homogeneous turbulent gray medium

- Zero mean 3D temperature field satisfying:
 
$$\langle \theta(\mathbf{r}')\theta(\mathbf{r} + \mathbf{r}') \rangle = \Theta^2 \exp(-\|\mathbf{r}\|/\Lambda)$$
- fine grid:  $320^3$ ,  $T_0 = 500$  K,  $\Theta = 50$  K
- $\kappa L = 25$ ,  $\kappa \Lambda = 1.25$



Dimensionless radiative power,  $x/L = 0.25$ ,  $N/\bar{N} = 4$



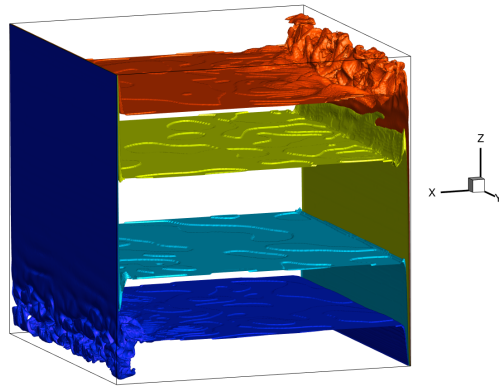
Effect of filtering level  
 $y/L = z/L = 0.5$



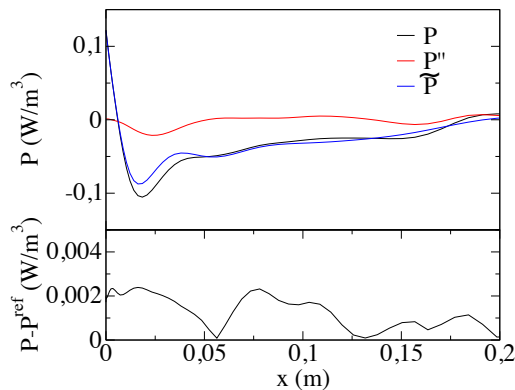
# Results

## Natural convection flow in a differentially heated cavity

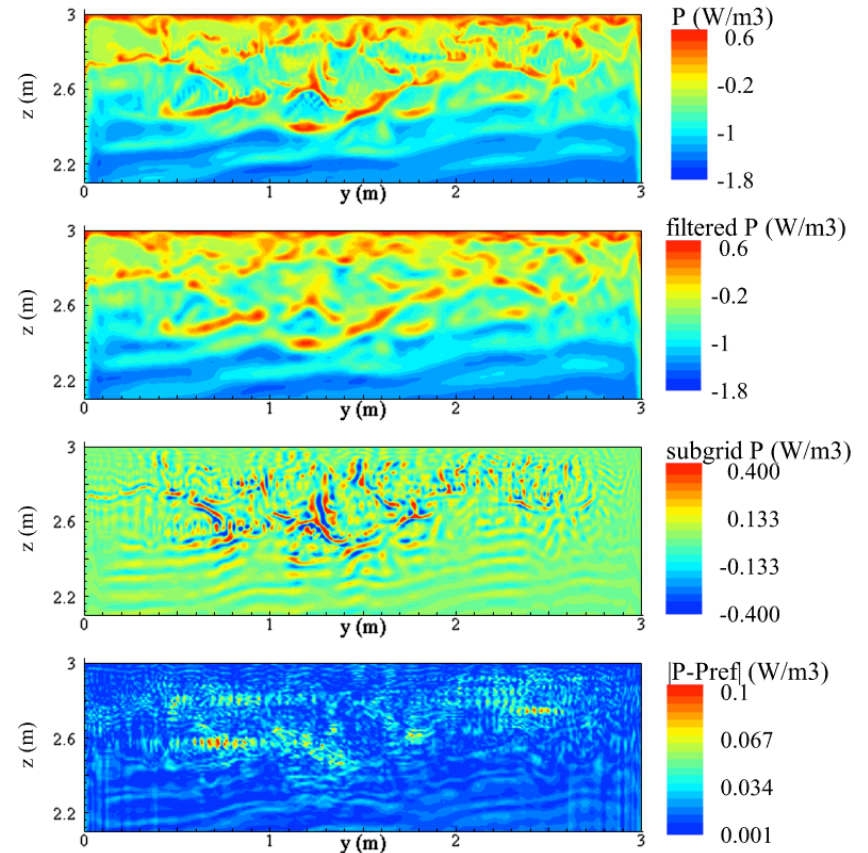
- fine grid:  $320^3$ , coarse grid:  $80^3$ ,  $Ra_L = 3 \times 10^9$ ,  $T_0 = 300$  K,  $L = 3$  m
- Non grey uniform absorption coefficient (global model)



instantaneous iso-temperature surfaces  
(without radiation)



radiative power profile near the hot wall

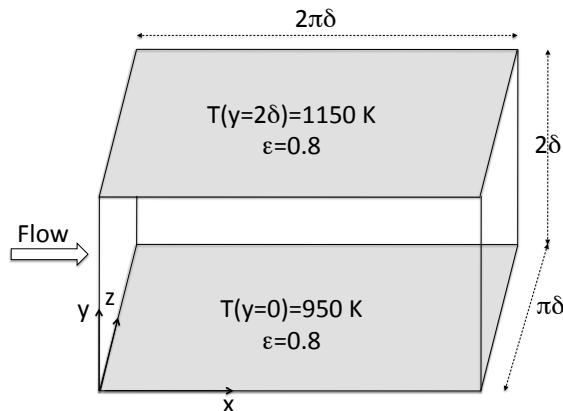


radiative power field  
in the B.L. near the hot wall

# Results

## Turbulent channel flow of hot combustion products

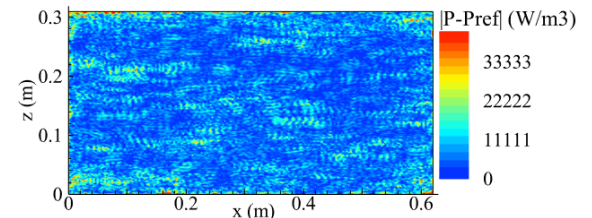
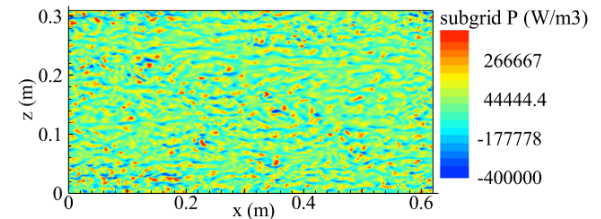
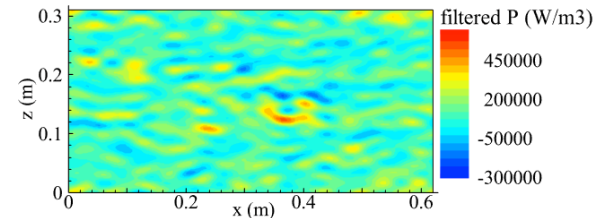
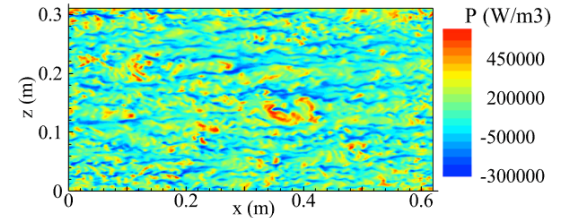
- Coupled calculations of Zhang et al. (2013)
- $Re_\delta = 1.175 \times 10^4$
- fine grid: 200x256x200
- coarse grid : 50x64x50



( $p=40$  atm,  $x(\text{H}_2\text{O}) = 0.155$ ,  $x(\text{CO}_2) = 0.116$ ,  $d = 10$  cm)

### Temperature dependent spectrum

- filtered contribution : sampling of the actual value of  $\kappa$  on the coarse mesh
- subgrid model : use of a uniform mean value



Radiative power field  
in the B.L. near the cold wall

## Motivations

### Radiative transfer model

- Filtering of the blackbody intensity

- Subgrid-scale model

- Implementation

### Results

- Homogeneous turbulent gray medium

- Natural convection in a differentially heated cavity

- Turbulent channel flow of hot combustion products

## Conclusion

# Conclusion

Soucasse *et al.*, JCP 257, 2014

## Strength

- Important CPU savings
- Accuracy (even near the walls and with temperature dependent spectra)
- Can be combined with other methods for the filtered part (Monte Carlo, DOM)

## Drawback

- Use of structured grids (Fourier decomposition)

## Perspectives

- How to combined this subgrid model with a LES of the flow ?